



The International Conference on
**Fluid and Thermal Energy
Conversion 2009**

December 7-10, 2009

Marine Bio-Education & Research Center
Gyeongsang National University
Tongyeong, Korea

Organized by

The Korean Society of Heat & Cold Energy Engineers
Gyeongsang National University
Eco-Friendly Heat & Cold Energy Mechanical Research Team of BK21
Institute of Marine Industry
Marine Bio-Education & Research Center

Faculty of Mechanical and Aerospace Engineering, Institut Teknologi Bandung
Mechanical Engineering Section, Persatuan Insinyur Indonesia
(The Institution of Engineers, Indonesia)
Society of Air Conditioning and Refrigeration Indonesia



Experimental Study on a Solar-Assisted High Temperature Heat Pump

Djuanda¹, Aryadi Suwono², Ari Darmawan Pasek², and Nathanael P. Tandian²

¹Makassar State University, INDONESIA

²Institute Technology of Bandung, INDONESIA

Contact Person:

Djuanda

Thermodynamic Laboratory, PAU Building 3rd Floor, ITB
Bandung, INDONESIA

Phone: +062 022 2502342, Fax: +062 022 2502342, E-mail: djuanda@students.itb.ac.id

Abstract

The performance of a solar-assisted high temperature heat pump for steam generation is described in this experiment. The heat pump utilizes a vapor compression cycle that consists of compressor, preheater, internal heat exchanger, evaporator, expansion device, and vertical steam generator. The main function of this system as a steam generation equipment needs a higher operating pressure than operating pressure of refrigeration cycles. Some limitation of this heat pump design are considered in this paper, primarily in availability of compressor. The design using a reciprocating compressor with maximum pressure of 24 bar. In this condition, the choice of refrigerant is an important factor. Isobutane (R-600a) is used in this study for some reasons. Although it is flammable, isobutane has low saturation pressure, ozone friendly, and low global warming potential. A preliminary test of this solar-assisted heat pump was done to obtain performance characteristic of high temperature heat pump sub-system. The preliminary test utilized steam from a boiler as a heat source. In the next experiment, heat from solar energy system will be used.

From the preliminary test, the output temperature of vapor could reach 120°C without the temperature of about 60°C. COP of the heat pump was 1.9 and maximum pressure ratio of the heat pump was 4.43. The water vapor flow out from steam generation was 1.1 kg/minute. In this condition, the primary function of the heat pump to generate steam was satisfied.

Keywords: heat pump, high temperature, solar energy, isobutane, heat pump performance.

1 INTRODUCTION

Concern for the environmental degradation will increase the utilization of clean energy sources. As solar energy, is receiving greater attention for various applications using different techniques. In the field of space and water heating, heat pump systems provide an efficient technology. This system also proved to contribute in the reduction of gases that cause global warming.

Heat pumps offer a technology that can use heat sources that are available from the environment such as air, soil, water, sun and other sources. By using less energy, heat pumps can increase the room and water temperature. Comparison of some water heating systems in terms of initial cost, operating cost, and efficiency of the system can be seen in Table 1 [6].

Table 1. Comparison of some water heating system

Types of water heater	Initial cost	Operating cost	Efficiency
Electric	Low	High	92%
Gas	Low	High	70%
Solar	High	Very Low	N.A
Heat Pump	Moderate	Low	300 – 600%

Heat pumps are also shown to reduce emission of CO₂. Data in 1997 showed the total global CO₂ emissions by 22 billion tons [5]. Heating room produces 30% of CO₂ emissions, while industrial sector producing emissions by 35%. With the use of heat pump can reduce 31% CO₂ emissions from the heating sector compared with the use of heating using oil-fired boilers. In 2002 total CO₂ emissions savings with heat pump is 1.2 billion tons, this means that approximately 6% of total CO₂ emissions. These emission reductions are the biggest reductions that can offer by a single technology. Further increase in power efficiency and heat pumps can reduce 16% CO₂ emissions.

Based on advantages in heat pumps compared with other systems, the development of heat pump technology as a heating system has good prospects. Heat pump technology found in cold season is more suitable for room and water heating, whereas for the tropics country such as Indonesia, development of heat pump is for water heating and drying systems.

Several types of heat pump are now available on the market, the vapor compression type using refrigerant such as R-22, R114, R-407 and R-134a. Because the system is mostly used for water heater, the heat pump ability is still limited primarily to the water out temperature was maximum 45°C [12]. Several systems have been tried like EVI cycle (*Enhanced Vapor Injection*) can produce output stream temperatures up to 90°C [14].

2 HIGH TEMPERATURE HEAT PUMP

Normally, heat flows from a higher temperature to lower temperature. By using a small amount of energy, flow direction can be reversed by using heat pump technology. This technology is useful to move heat from natural sources such as air, water, soil, waste heat or solar energy into a building used by the commercial and industry sector.

There are two main types of heat pumps, the vapor compression heat pump and absorption heat pumps. Generally, the heat pumping can be achieved through a variety of thermodynamic cycles and processes such as the Stirling Cycle, Vuilleumier Cycle, sorption solid-vapor system, hybrid systems and magnetic processes. In consideration of the higher COP, this research is focused on the development of vapor compression heat pump.

Development of vapor compression heat pumps started to be developed in the late 19th century and began to be widespread at the beginning of the 20th century. The main components of this cycle are the compressor, expansion device and two heat exchangers, each of which serves as the evaporator and condenser. In a few CO₂ heat pump systems, a condenser function is replaced by cooler gas.

High temperature heat pump is used utilizing solar collector system as a source of heat and steam generators to replace the function of a condenser. The scheme of this system is shown in Figure 1. The system is divided into two parts, namely the sub-solar heating system that is used as a heat source for heat pumps, and the second is a heat pump sub-system.

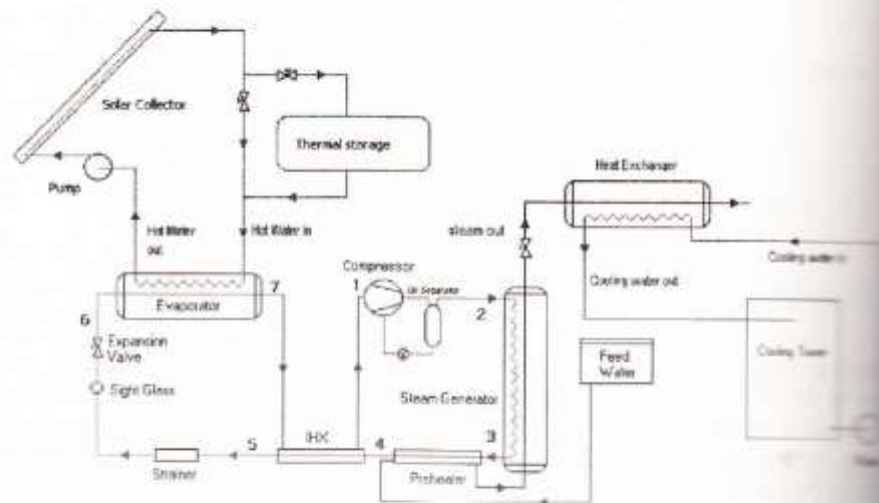


Figure 1 High temperature heat pump assisted solar energy heating

Evaporator transfer energy from the heat source of the sub-solar heating systems to sub-cooled liquid refrigerant. The flow of refrigerant from the evaporator through an internal heat exchanger and is compressed in the compressor to high pressure and temperature. Refrigerant then flows through the condenser to steam generators, and then transferred to the flow of water that flows through the steam generators. The flow of water will turn it into steam flow. Before entering the steam generator, water flows through a pre-heater which will increase the initial temperature of the water from the initial conditions.

Several studies using heat pumps in combination with other heat sources has also been reported. Odeh et al. [10] did research on the combination of heat pump which uses solar energy as a heat source. The solar collector is used as a multiplier effect on the evaporator. Refrigerant used is R-134a and the water is used as the working fluid in solar energy systems. Double effect evaporator is obtained by combining heat obtained from the solar collector and the heat released by the condenser. Increasing water flow rate in the evaporator will increase the heat released by the condenser as well as an increase input water temperature will increase the heat released by the condenser and COP of the heat pump.

Heat pump combined with solar energy used by Yumrutas and Kaska [15] for the purpose of space heating. The system using refrigerant R-22 and thermal storage using water as working fluid. The COP_s is about 2.5 for a lower storage temperature at the end of a cloudy day and it is about 3.0 for a higher storage temperature at the end of a sunny day, and it fluctuates between these values in other days. Also, COP_s turns out to be about 15–20% lower than COP_{HP}.

Chaichana et al. [16] compared the performance of refrigerant R-22 to the hydrocarbon refrigerants (R-290, R-600, R-600a, R-1270), R-717 and R-744 in the solar-boostered heat pump. Analysis shows that the R-717 has more advantages when compared with other refrigerant. Regardless of the material, R-717 can use R-22 compressor. Moreover, with a high latent heat, the charge amount charged is also smaller. The weakness of R-717 lies in its toxic, corrosive to copper and the operating temperature reach 200°C, which may result in the decomposition of lubricating oil.

can be directly substituted into the R-22 systems with little modification for safety considerations, and the COP system will be slightly reduced compared to R-22.

Research [4] researched the various refrigerants as a substitute for CFC refrigerant, refrigerant HFE-134 promising for use in high temperature applications as a substitute for CFC-114. HCFC-123 used for replacement CFC-11, while HFC-245ca and HFC-143 is used as a substitute for CFC-11 for cooling term.

From various studies that heat pumps currently available, the presence of vapor compression heat pumps from 1985 to 2008 can be displayed in a diagram form as in Figure 2. This diagram shows the various types of refrigerant used in heat pumps so far and achieved a condenser temperature both pure refrigerant and refrigerant mixtures. The maximum temperature of the condenser heat pump is located within the range of 90°C.

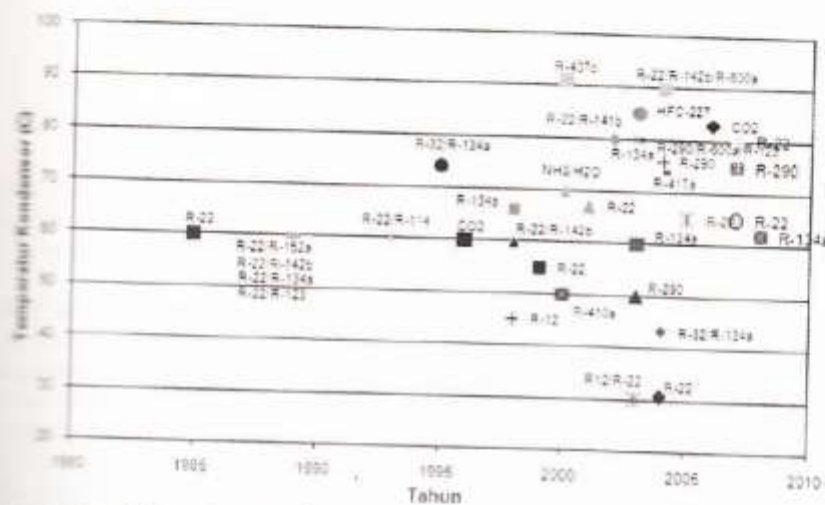


Figure 2. Diagram of the existence of vapor compression heat pump from the year 1985 - 2008

Results of the study also showed that the presence of heat pumps as a producer of steam has not been a variety of benefits that have heat pumps and solar energy combined with a heat source at the same time, the heat pump as the assessment of steam-producing potential to be developed.

4 EXPERIMENTAL RESULT

Steam is passed into heat exchanger so that heat input to the evaporator can be done by using isobutane (R-600a). Control of hot water flow in the evaporator is

From the temperature graphs that occur in Figure 3, it can be seen that the condenser temperature increases when the evaporator is given heat from the steam generator. The maximum temperature of the refrigerant reaches 140°C after 95 minutes of run time. At the same time, the water in the steam generator is also experiencing a phase change from liquid to steam at the temperature source 75°C. The maximum steam temperature generated reaches 120°C with a flow rate that reaches 0.11 kg/min. This sub-system reaches equilibrium in accordance with the conditions of temperature refrigerant into steam.

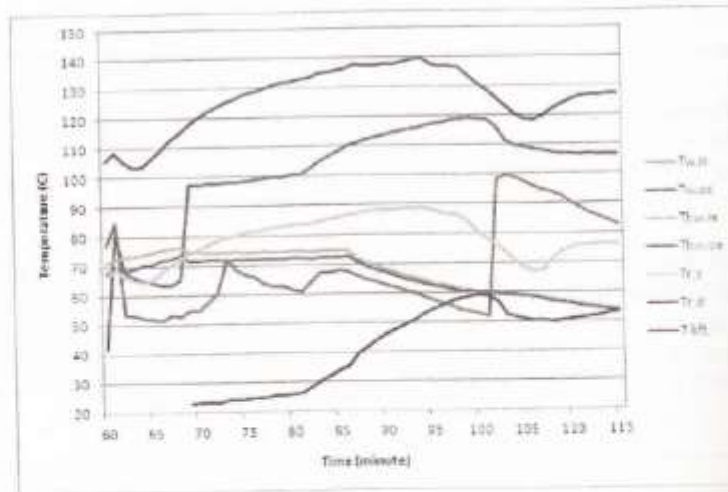


Figure 3 Temperature history of heat pump

Another point of concern is lift temperature generated by the heat pump system. Lift temperature is defined as the temperature difference between the output generated steam generators and pressure in the evaporator ($T_{r,os} - T_{w,je}$). The results of testing on the system can be seen in Figure 3.

When given the heat from the evaporator boiler, lift temperature will decrease. This is because the increase in water temperature input to the evaporator while the temperature of water in the steam generator is not a comparable increase. At the time of steam began to form in the steam generator, there is an increase of very high temperature. The maximum lift temperature that can be achieved is about 50°C, then decreased with increasing temperature input in the evaporator and then water temperature of about 50°C.

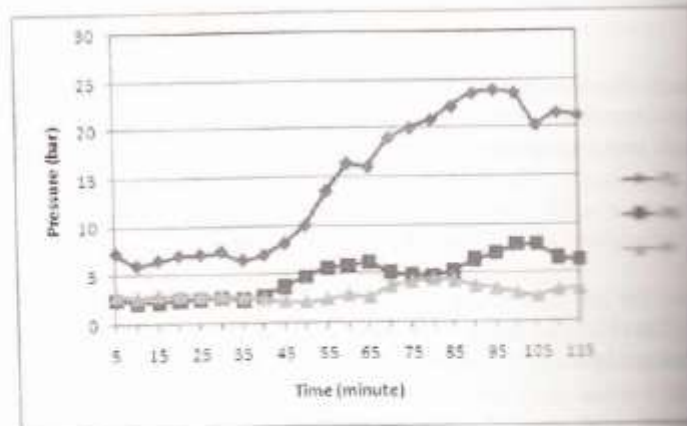


Figure 4 Pressure and ratio of pressure of the heat pump

Increasing the temperature of the system itself also affects the pressure. The pressure on the steam generator also experienced significant improvement with increasing evaporator heat. The maximum pressure achieved by the steam generator reaches 25 bar, respectively in the evaporator. Pressure ratio which can be obtained from the test results is

Experimental Study on a Solar-Assisted High Temperature Heat Pump

As can be seen from Figure 4. The ratio of the maximum pressure that can be achieved by a heat pump system using isobutane reaches 4.43, whereas the minimum ratio reached 2.13.

By using boiler for heating source, simulation of solar assisted high temperature heat pump was done. The steam from boiler flow into the heat exchanger firstly and then transferred to the flow of water into the evaporator. The amount of heat input to the evaporator is set by adjusting the mass flow into the evaporator. Settings needed in order to simulate the heat input from solar collector. Variations in heat input to the evaporator and the system performance can be seen in Figure 5.

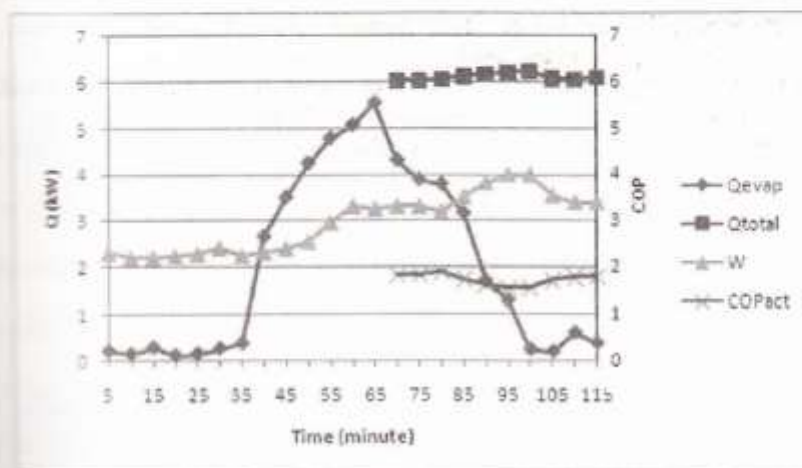


Figure 5 Performance of heat pump

variation in heat input to the evaporator was like sine graph. Where there is an increase in heat until reaches the peak point and then declined again. Maximum heat input to the evaporator reaches 5.6 kW from the graph it can also be seen that along with increasing the heat input, compressor power also increasing and reached a maximum of 4 kW. While the maximum heat output of the steam compressor and pre-heater reaches 6.24 kW. With the output power would be obtained for the COP around 4.4. The low COP of the system, because still many losses that occurred in the heat pump system.

5 CONCLUSION

Isobutane refrigerant has been used as a working fluid at high temperature heat pump. By providing heat input to the evaporator in order to change the lift temperature heat pump can be increased up to 300%.

However, the compressor output pressure also increased reaching 300% compared to before the increase in the evaporator. And the ratio of the maximum pressure achieved by the heat pump to isobutane. From this research, isobutane is potential refrigerant for heat pump application at high temperature.

ACKNOWLEDGMENTS

This research is supported by Research and Social Service (LPPM) ITB for the support of funds provided by Research Program (KK) 2008 ITB.

REFERENCES

- [1] Brunin, O., M. Feidt and B. Hivet. "Comparison of the working domains of some compression heat pumps and a compression-absorption heat pump", *Int. J. Refrigeration*, **20:5**, 398 – 406, 1997.
- [2] Coelho, L.M.R., *Basics of Heat Pump Technology*, Workshop Groundhit, Brussel, 2006.
- [3] Departemen Energi dan Sumber Daya Mineral. *Kebijakan Pengembangan Energi Terbarukan dan Konservasi Energi (Energi Hijau)*, Jakarta, 2003.
- [4] Devotta, S. "Alternatif heat pump Working fluids to CFCs", *Heat Recovery Systems & CHP*, **15:3**, 273-279, 1995.
- [5] Halozan, Hermann, Gilli, Paul Viktor, "Heat pump for different world region-Now, and with future", *18th World Energy Congress*, 2002.
- [6] Hawaiian Electric Company Inc. *Guide to Heat Pump Water Heating for Condominiums, Commercial, and Institutional Facilities*, Energy Services Department, 2003.
- [7] IEA Heat Pump Centre. "Norwegian heat pump sales boom", *IEA Heat Pump Centre Newsletter*, **1:4**, 7, 2003.
- [8] Kenisarin, M., Khamid, M. "Solar energy storage using phase change materials", *Renewable and Sustainable Energy Review*, 2006.
- [9] Narodoslowsky, M., F. Windisch, F. Mose. "New compression heat pump media for medium-high temperature application", *Heat Recovery system & CHP*, **8:1**, 23-31, 1998.
- [10] Odeh, S., Saleem N., Bilal A. "Performance evaluation of solar-assisted double-tube evaporator heat pump system", *Int. Comm. Heat Mass Transfer*, **31:2**, 191 – 201, 2004.
- [11] Sarkar, J., Bhattacharyya, S., Gopal, M.R. "Natural refrigerant-based subcritical and transcritical cycles for high temperature heating", *International Journal of Refrigeration*, 1-8, 2006.
- [12] Smith, F.J., Meyer J.P. *Potential of Non-Zeotropic Refrigerant Mixture as Working Substances in Hot Water Heat Pumps*, Research Group for Heating and Cooling Technology, Laboratory of Energy, Auckland Park, South Africa. 2006.
- [13] Ure, Z. "Positive temperature eutectic (PCM) thermal energy storage systems", *International Congress of Refrigeration*, Washington DC. ICR0606, 2003.
- [14] Viessmann. *Technical Series Heat Pumps*, 2004.
- [15] Yumrutas, R., Onder, K., "Experimental investigation of thermal performance of a solar-assisted heat pump system with a thermal storage", *International of Journal Energy Research*, **28**, 169-175, 2004.
- [16] Chaichana, C., Aye L., Charters W.W.S., "Natural Working Fluids for Solar-assisted heat pumps", *International Journal of Refrigeration*, **26**, 1-8, 2003.